

# **FORCE-BASED ASSESSMENT OF WELD GEOMETRY**

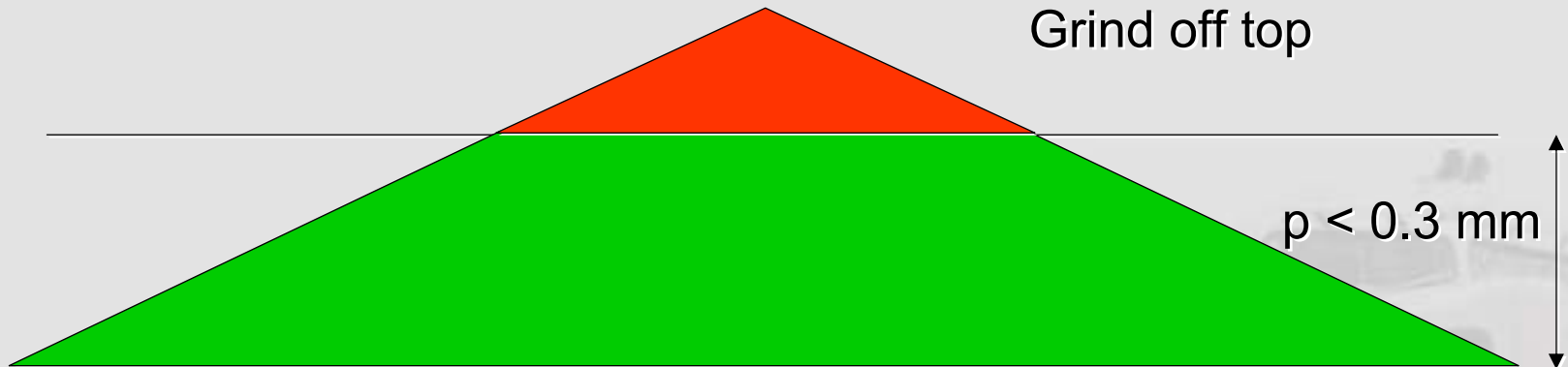
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# DAMAGE DUE TO POOR WELD GEOMETRY



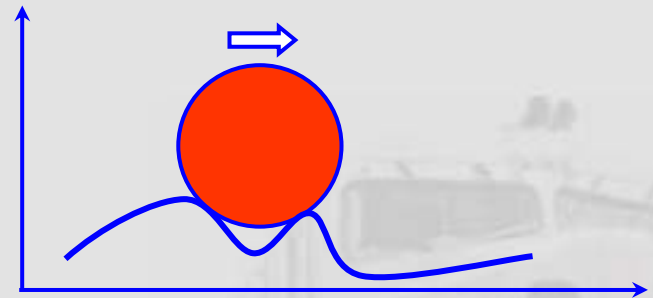
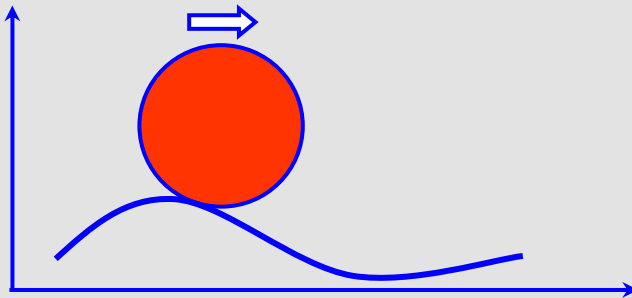
# EXISTING WELD GEOMETRY STANDARDS



For example

Versine:  $0 < p < 0.3 \text{ mm}$

# IMPACT LOAD: 1 AND 2-POINT CONTACT



- 2-point contact only relevant for joints. Under normal conditions jump is 10-20  $\mu\text{m}$  and disappears by plastification
- Study concentrated on 1-point contact

## MODEL ASSUMPTIONS (1)

- Weld geometry is sampled discretely
- Wavelengths shorter than 25 mm are filtered out
- Maximum wavelength is 2 m (1 m straightedge)
- Wavelength range 10 cm – 2 m

## MODEL ASSUMPTIONS (2)

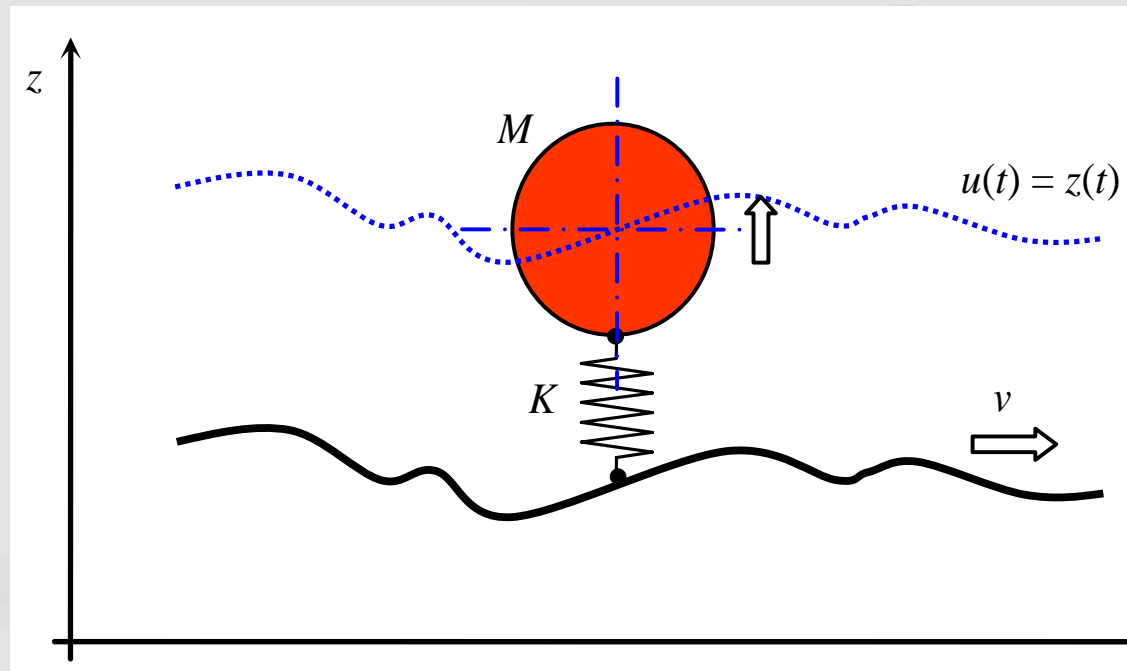
- Speed range: 40-300 km/h
- Frequency range 10-2000 Hz.
- For the development of a practical tool it was decided to consider just one constant mass-stiffness combination.

# ACCELERATION APPROACH

- Wheel follows rail irregularities;
- Dynamic part of contact force is governed by:

$$F_{dyn}(t) = M\ddot{z}(t)$$

$$F_{dyn} = \alpha Mv^2 \frac{d^2 z}{dx^2}$$



## VELOCITY APPROACH (1)

Assumption:

Equivalent wheel mass is proportional to wavelength:

$$F_{dyn}(t) = M_e \ddot{z}(t)$$

with:

$$M_e = \frac{1}{L_0} ML = \frac{1}{L_0} M \frac{v}{f} \quad \text{and:} \quad \ddot{z} = \dot{z} \frac{2\pi v}{L}$$

## VELOCITY APPROACH (2)

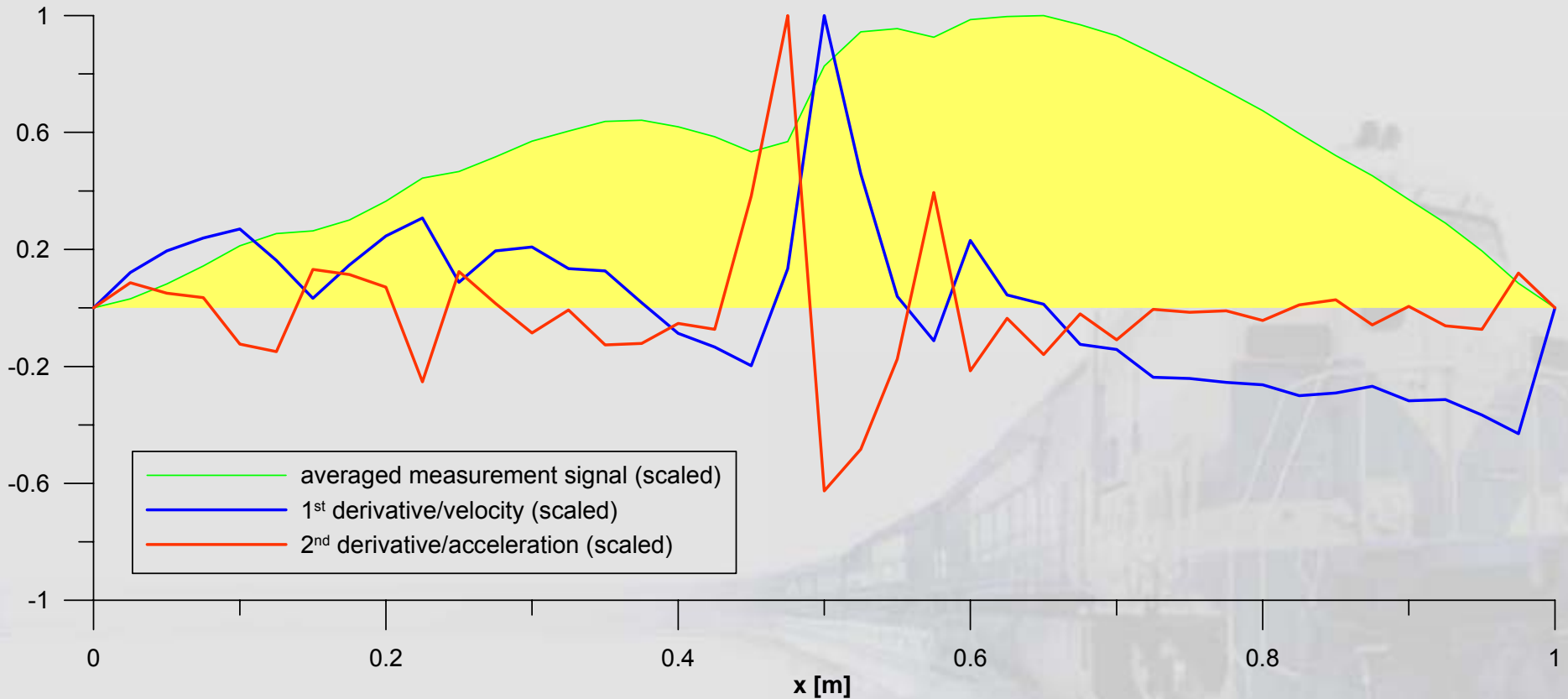
The dynamic contact force as a function of the first time derivative:

$$F_{dyn} = M \frac{2\pi v}{L_0} \dot{z}$$

The dynamic contact force in terms of the spatial derivative, including calibration factor  $\beta$ :

$$F_{dyn} = \beta \frac{M}{L_0} v^2 \frac{dz}{dx}$$

# ACCELERATION VERSUS VELOCITY



# EVALUATION BASED ON VELOCITY

Force intervention values (relative) and inclinations

<b>Velocity</b>	<b>Force</b>	<b>Inclination</b>
40 km/h	5 kN	3.2 mrad
80 km/h	15 kN	2.4 mrad
140 km/h	35 kN	1.8 mrad
200 km/h	65 kN	0.9 mrad

# COMPARISON OLD & NEW STANDARDS

Old standard ( $0 < p < 0.3$  mm): 76% rejected

New standard:

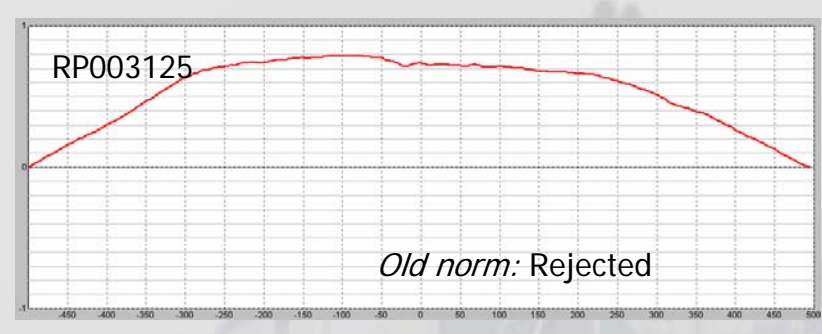
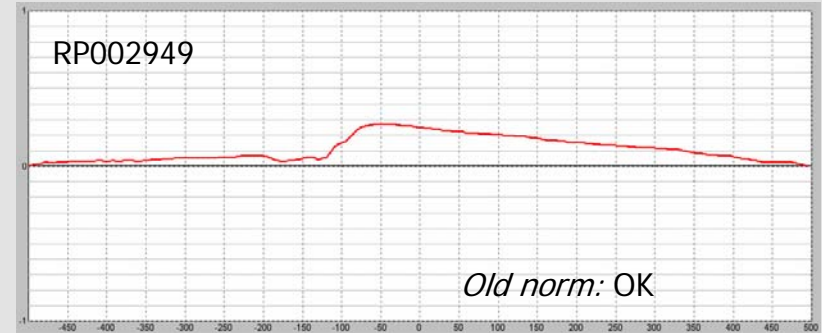
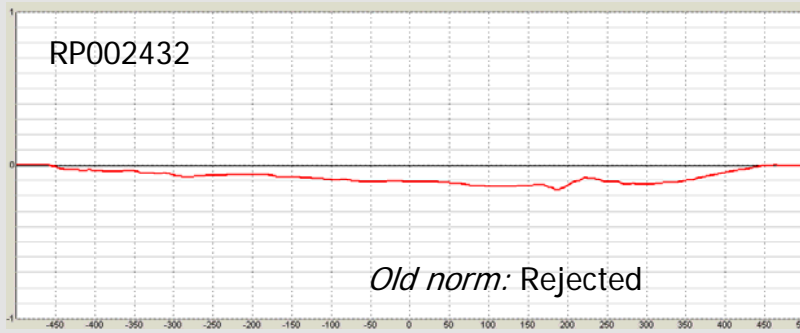
- 40 km/h: 33% (acc. 33%)
- 80 km/h: 53% (acc. 64%)
- 140 km/h: 81% (acc. 93%)
- 200 km/h: 86% (acc. 94%)

## QUALITY INDICES (QI)

$$QI = \frac{|F_{max, actual}|}{F_{norm}} \leq 1 \quad \Rightarrow \quad OK$$

- $QI \leq 1$ : Accepted
- $QI > 1$ : Rejected

# ASSESSMENT OLD AND NEW



NORM BASED ON ACCELERATIONS					NORM BASED ON VELOCITIES			
[km/h]	norm 40	norm 80	norm 140	norm 200	norm 40	norm 80	norm 140	norm 200
[kN]	30	80	150	300	5	15	35	65
	SCORE PER NORM: (<1: OK)				SCORE PER NORM: (<1: OK)			
RP002432	1.3	1.7	3.3	3.3	0.5	0.7	0.9	1
RP002945	1.3	1.7	3.3	3.3	0.6	0.8	1	1.1
RP002949	1.1	1.7	2.5	2.5	1.1	1.7	2	2.5
RP003125	0.6	0.9	1.4	1.4	1.1	1.4	2	2

# EXTENSION TO HEAVY HAUL AND HSL (1)

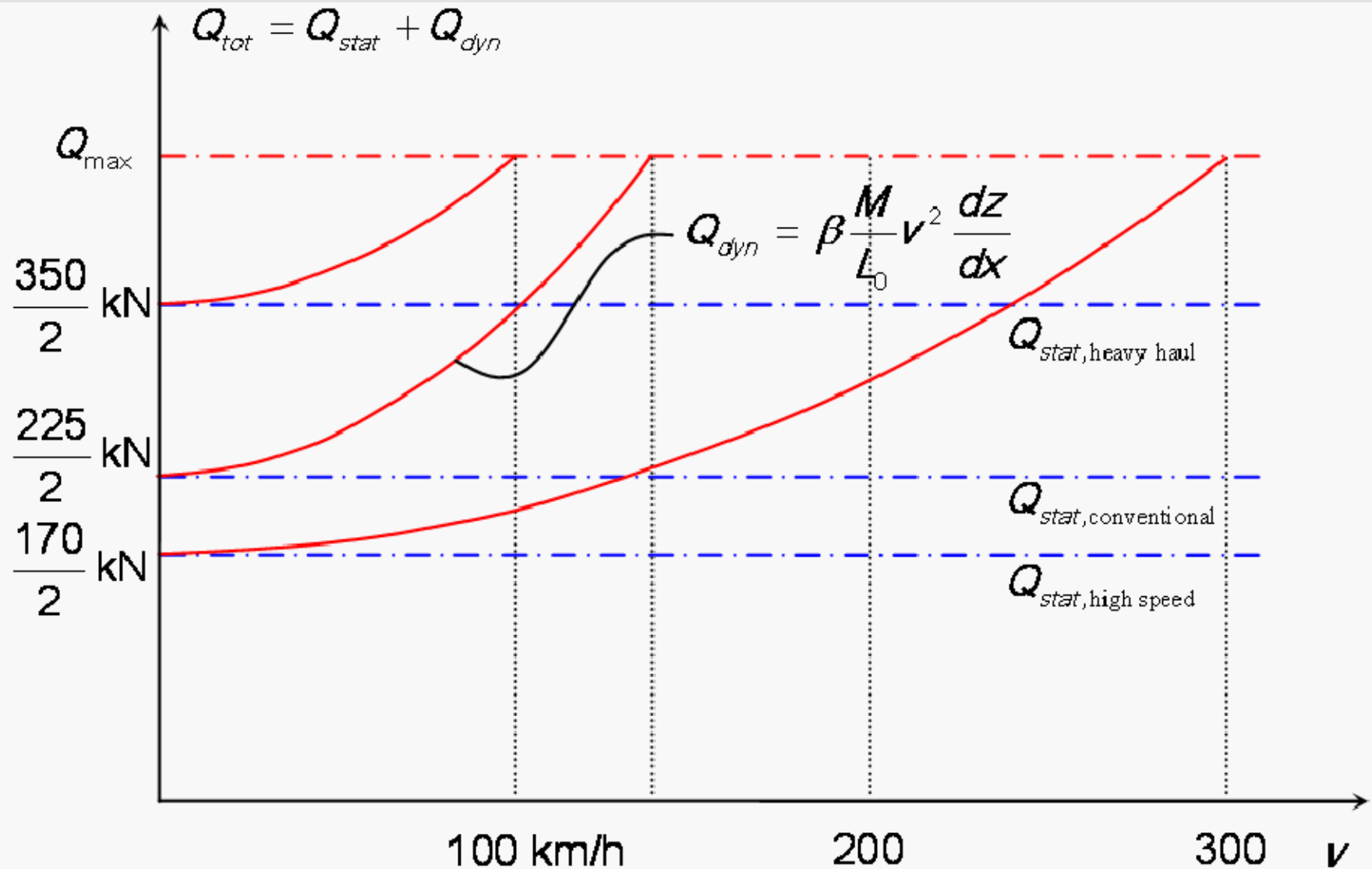
Total wheel load versus velocity:

$$Q_{tot} = Q_{stat} + \beta \frac{M}{L_0} v^2 \frac{dz}{dx} \Rightarrow$$

$$\frac{dz}{dx} < \Delta Q \frac{1}{\beta} \frac{L_0}{M v^2}$$

$Q_{max}$  is approximately 450/2 kN

# EXTENSION TO HEAVY HAUL AND HSL (2)



## EXTENSION TO HEAVY HAUL AND HSL (3)

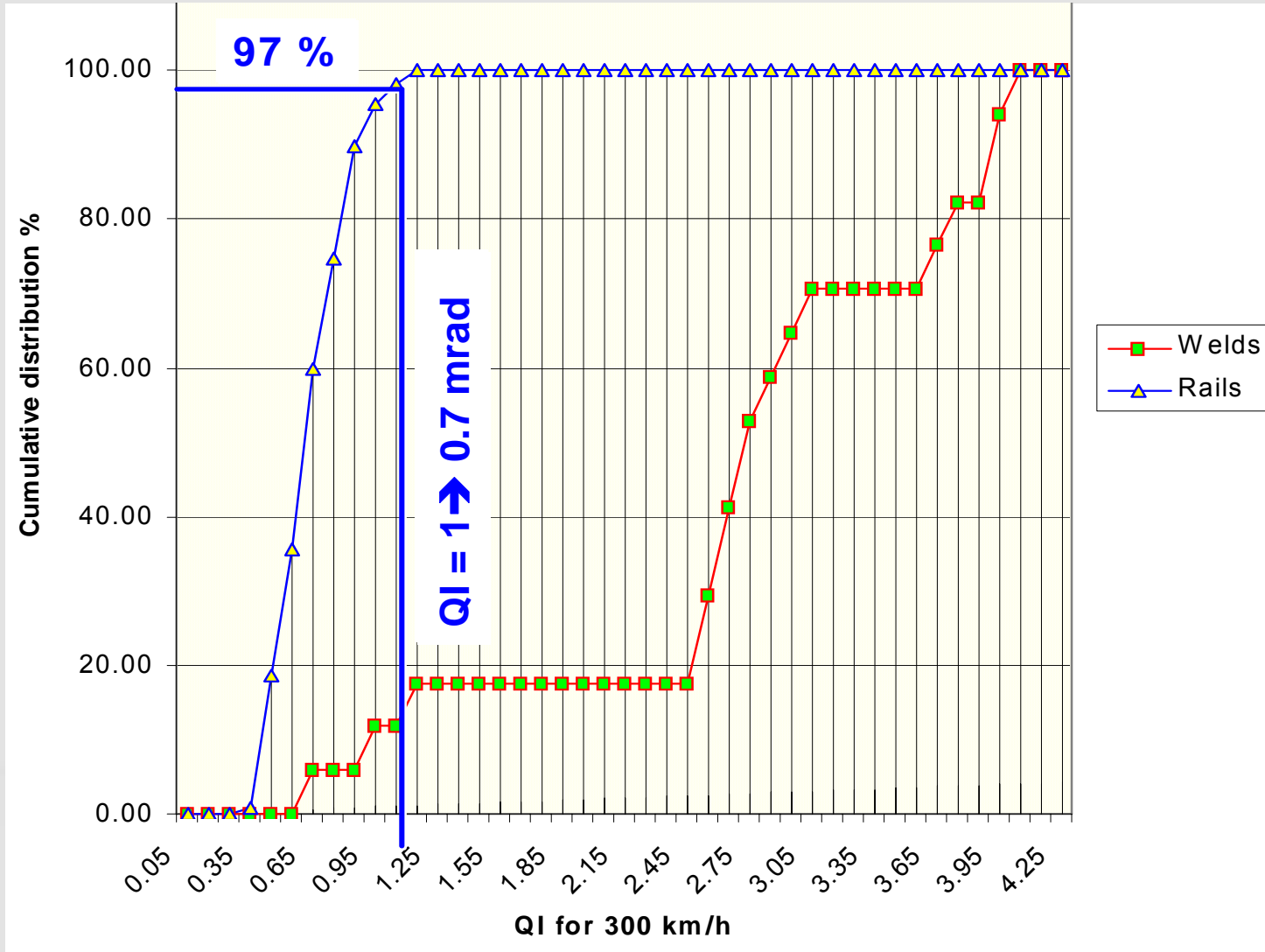
Intervention values for Heavy Haul and HSL lines:

	[kN] $\Delta Q$	$v$ [m/s]	$\frac{dz}{dx} \cdot \frac{\beta M}{L_0}$	Norm value [mrad]
<i>Conventional</i>	225/2	40	0.070	1.8
<i>Heavy Haul</i>	100/2	30	0.056	1.4
<i>High-Speed</i>	280/2	85	0.019	0.5

## EXTENSION TO HEAVY HAUL AND HSL (4)

- The value of 0.5 mrad as max. inclination for HSL was changed to 0.7 mrad based on 100 measurements of new HSL rails;
- 97 % of rails is better than 0.7 mrad → QI = 1;
- Standard can only be achieved by applying grinding train (Plasser GWM).

# RAIL & WELD GEOMETRY HSL

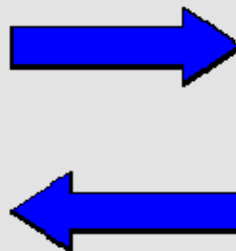


# PRACTICAL IMPLEMENTATION (1)

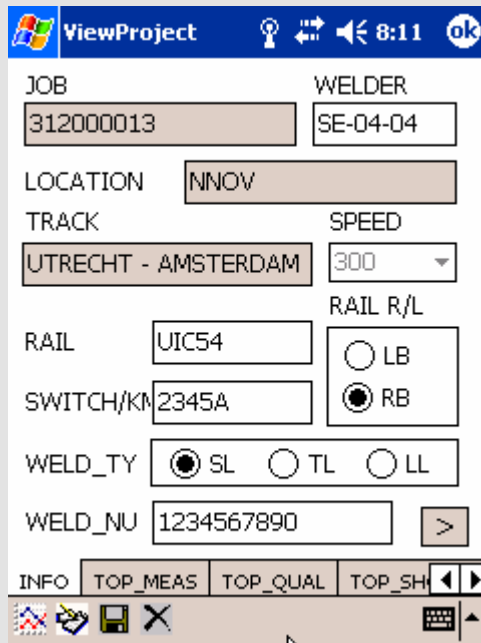
Procedure:

- Sample weld geometry with digital straightedge
- Filter measured signal
- Determine 1<sup>st</sup> derivative (inclination)
- Normalize with intervention value for line speed
- Calculate QI.
- $QI < 1$ : OK, otherwise: grinding.

# PRACTICAL IMPLEMENTATION (2)



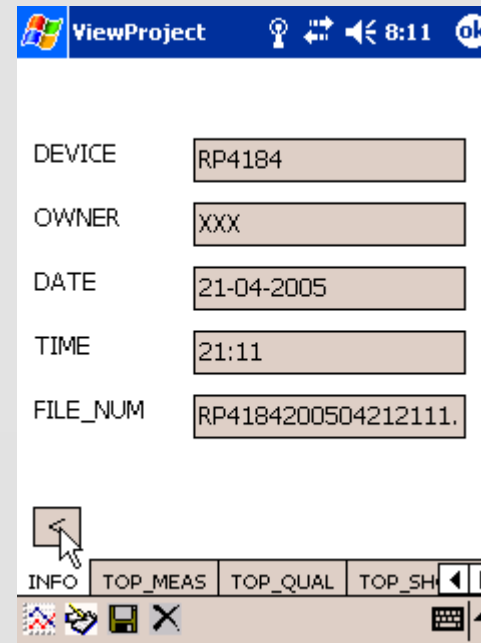
# EXAMPLES OF PDA SCREENS (1)



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312000013	SE-04-04
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UIC54	<input type="radio"/> LB
SWITCH/KM 2345A	<input checked="" type="radio"/> RB
WELD_TY	<input checked="" type="radio"/> SL <input type="radio"/> TL <input type="radio"/> LL
WELD_NU	1234567890 >

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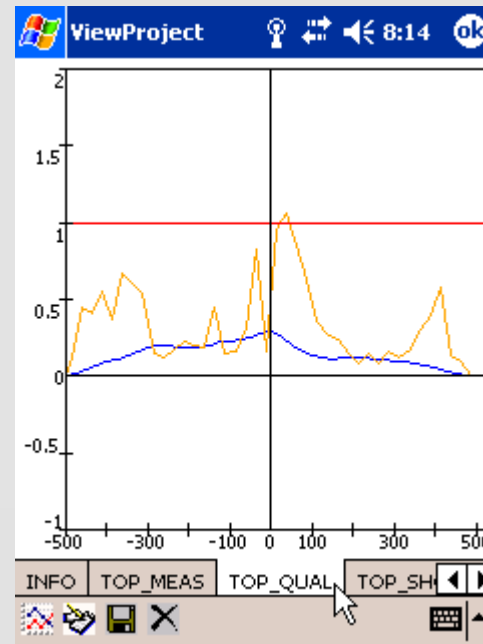
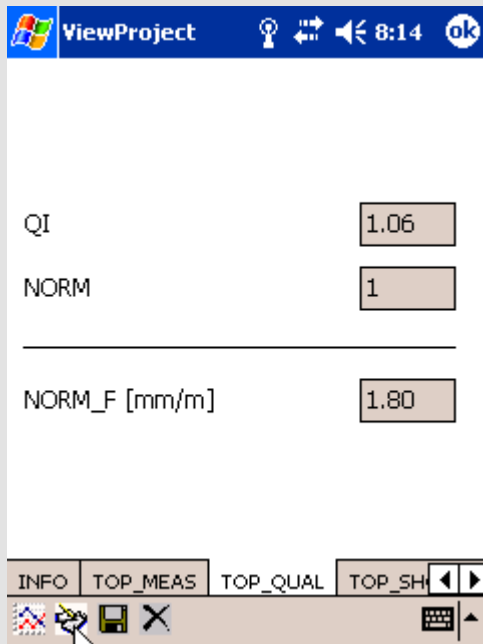


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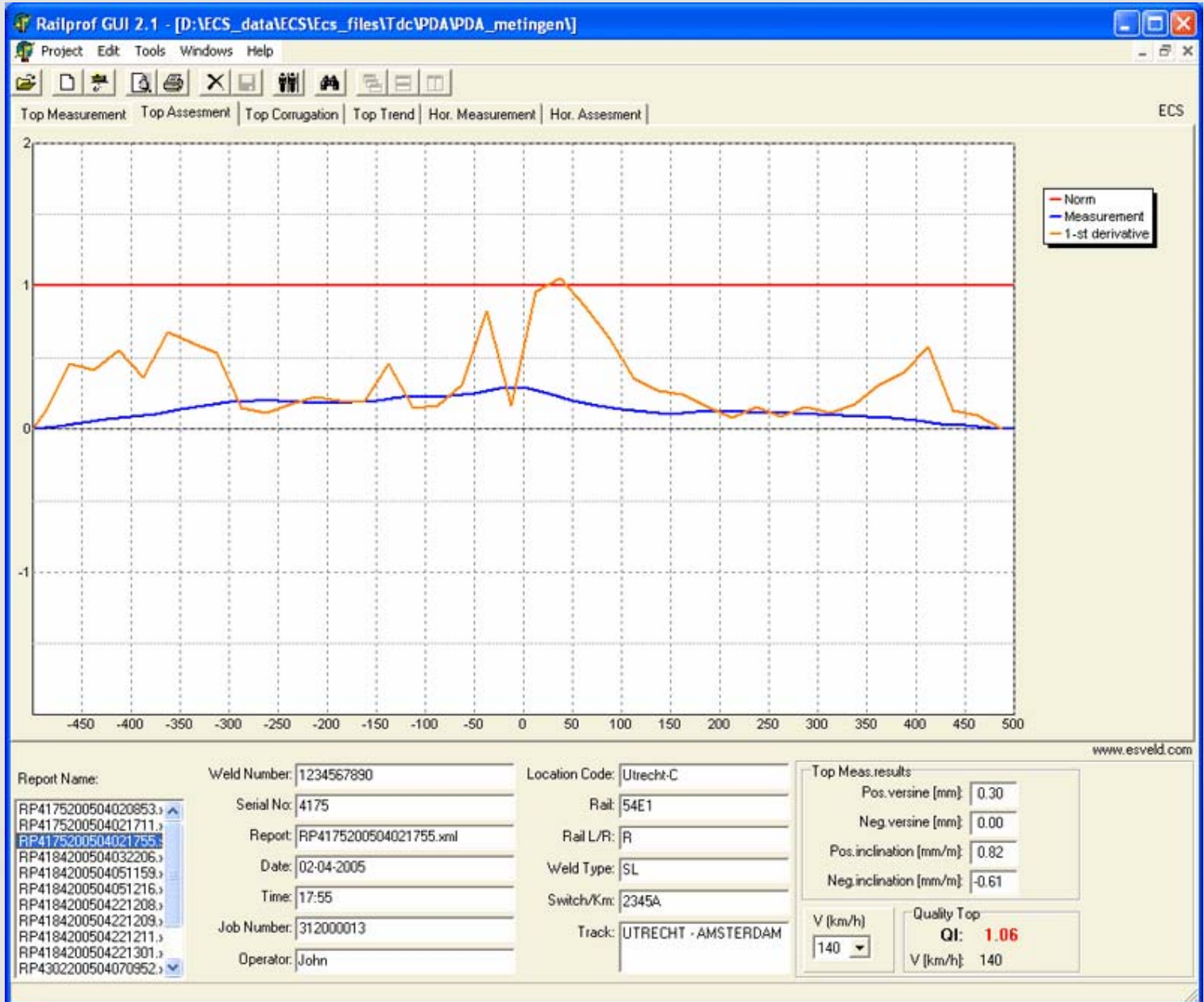
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# EXAMPLES OF PDA SCREENS (2)



$V = 140 \text{ km/h}$

DESKTOP SOFTWARE



## CONCLUSIONS

1. The theory developed on the basis of the first derivative is powerful and practical;
2. New, high quality rails have a first derivative  $< 0.7$  mrad;
3. Mechanical grinding (GWM) is inevitable to achieve such an accuracy for weld geometry;
4. Steel straightedges are absolutely inadequate;
5. Instead accurate electronic straightedges (RAILPROF), producing versine and 1<sup>st</sup> derivative, are required;
6. The presented concept is very well applicable to heavy haul tracks.